

Comparison of parameterization schemes for solving the discrete material optimization problem of composite structures

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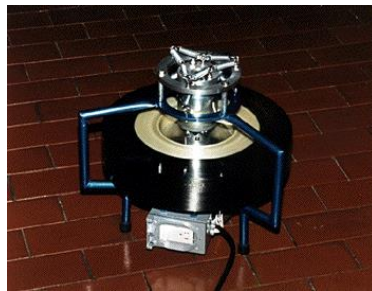
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INTRODUCTION

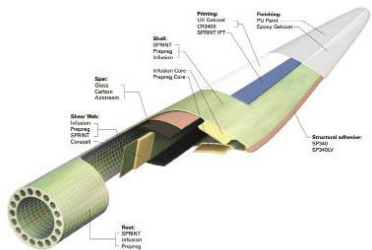
- Development of new renewable energy systems: high performance materials (strength, durability...)



- Sustainability of transportation systems: lightweight solutions



- Revived interest in composite structures.
➔ optimization of composites to take the best of their performances



INTRODUCTION

- SS Great Britain (1843)
 - First ship to be built with an iron hull
 - Parts designed and fixed together according to the available technology at the time (wood technology)
 - If you don't know it is metal you would think it is wood!

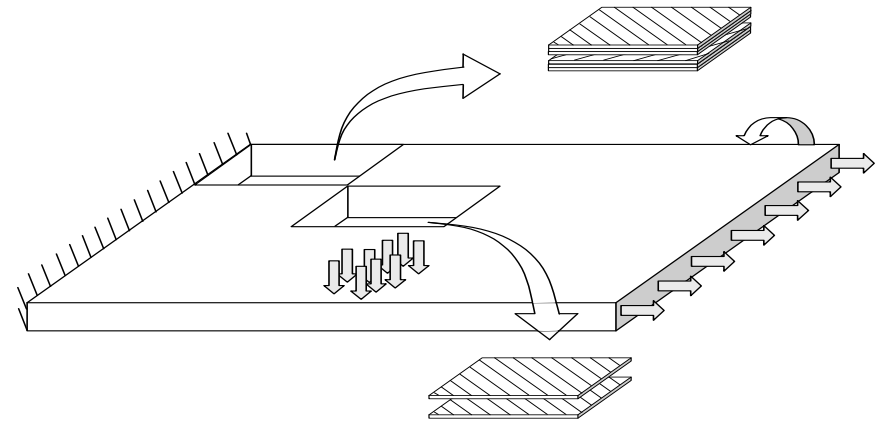
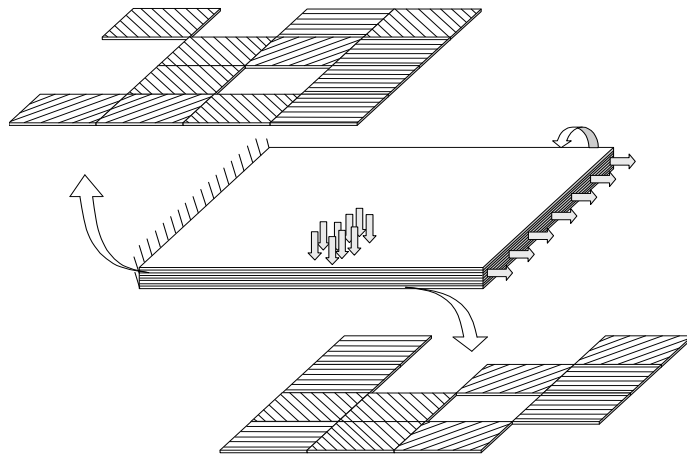


INTRODUCTION

- Boeing 787
 - Composite structure BUT...
 - True potential is not fully explored
 - Composite structures are almost direct replicas of metallic structures
 - Some of the issues
 - Repair
 - Failure modes
 - Systems interaction
 - Weight saving $\leq 5\%$ in current use



- Classical design problems of composite structures to be addressed:
 - Optimal layout of laminates over the structure
 - Through-the thickness-optimization of composites: stacking sequence optimization



- General/global approach to address simultaneously optimal layout and stacking sequence → **Discrete Material Optimization** approach (Lund & Stegmann, 2005)

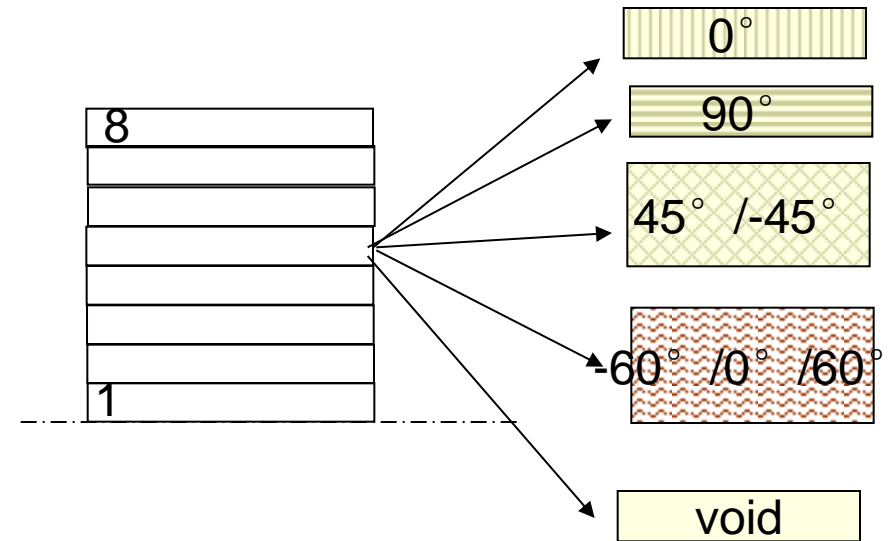
Discrete Material Optimization approach

- Formulate the optimization problem as a 'n' materials selection problem
- Use an extended topology optimization approach to solve the problem in continuous variables
 - Interpolation /parameterization scheme of material properties
 - Continuous optimization problem with penalization of intermediate solutions

$$\mathbf{C}_i = \sum_{j=1}^m w_{ij} \mathbf{C}_i^{(j)}$$

$$0 \leq w_{ij} \leq 1 \quad \sum_{j=1}^m w_{ij} \leq 1$$

$$w_{ik} = 0 \quad (k \neq j) \text{ when } w_{ij} = 1$$



- Topology optimization of laminate
 - Selection of ply orientation and layout of the laminate
 - Add one an existence (density variable) μ in $[0,1]$

$$c^l = (\mu_l)^q \left(\sum_{i=1}^{n^l} w_i^l c_i^l \right)$$

- Introduce a volume constraint of the foam or of the fiber material

$$\sum_{l=1}^{n_v} \mu_l V_l \leq \bar{V}$$



■ This work:

- To compare DMO, SFP and BCP
- To investigate the approach parameters such as the penalization
- To tailor a robust solution procedure based on the sequential convex programming
- To validate the work with applications
 - Academic examples
 - Larger scale problems with real-life characteristics

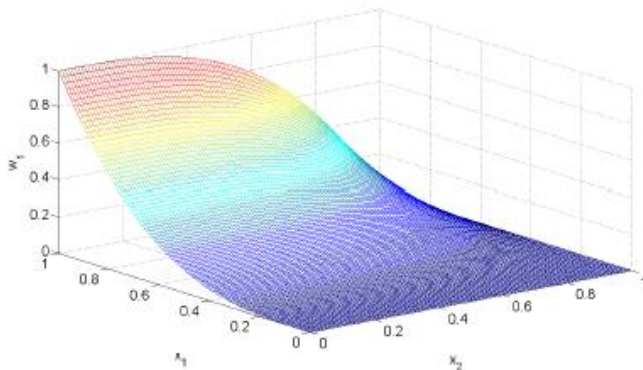


- DMO4 interpolation scheme:
 - Extension of Thomsen (1992) and Sigmund & Torquato (2000) topology optimization schemes

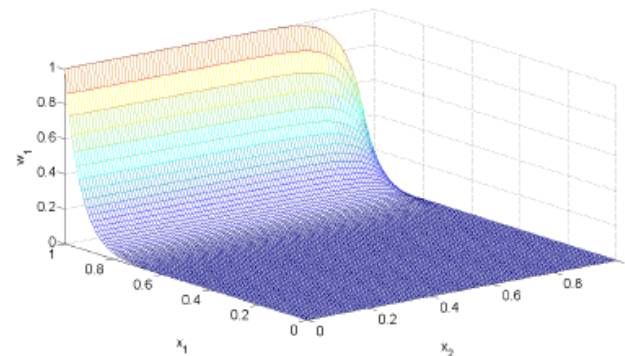
$$E = E_1 + x^p (E_2 - E_1)$$

- Introduces one existence variable ($[0,1]$) per material
- Uses a power law (SIMP) penalization of intermediate densities

$$w_{ij} = x_{ij}^p \prod_{\xi=1}^{m_v} (1 - x_{i\xi}^p) \quad \text{with} \quad 0 \leq x_{ij} \leq 1$$



w_1 with $p=3$



w_1 with $p=15$

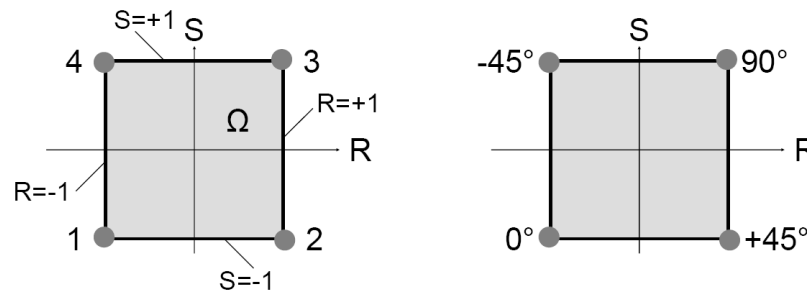
Shape Function with Penalization (SFP)

Bruyneel (2011)

- SFP scheme makes use of the Lagrange polynomial interpolation of finite element shape functions
 - For $0^\circ / 90^\circ / 45^\circ / -45^\circ$: four-node finite element

$$w_1 = \frac{1}{4}(1-R)(1-S) \quad w_2 = \frac{1}{4}(1+R)(1-S)$$

$$w_3 = \frac{1}{4}(1+R)(1+S) \quad w_4 = \frac{1}{4}(1-R)(1+S)$$



- Introduces a power penalization (SIMP)

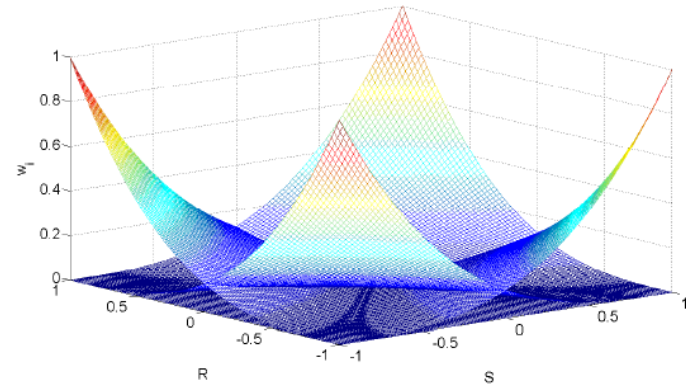
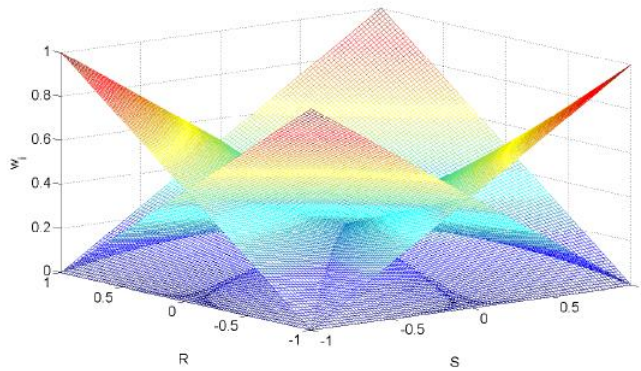
$$w_i^{SFP} = \left[\frac{1}{4}(1 \pm R)(1 \pm S) \right]^p$$



Shape Function with Penalization (SFP)

Bruyneel (2011)

■ SFP shape functions and penalization



$$w_i^{SFP} = \left[\frac{1}{4} (1 \pm R)(1 \pm S) \right]^p$$

- Two design (instead of 4) variables ranging in $[-1,1]$
- Extension to 'n' node finite elements is theoretically possible, but problem rapidly complex in practice

Bi-value coded parameterization

(G. Tong et al. 2011)

- Bi-value coding parameterization generalizes the SFP scheme
- Abandon the shape function idea, but keep the idea of coding the materials using bi-value variables (typically $[-1,1]$)

$$w_{ij} = \left[\frac{1}{2^{m_v}} \cdot \prod_{k=1}^{m_v} (1 + s_{jk} x_{ik}) \right]^p \quad \text{with } -1 \leq x_{ik} \leq 1 \text{ and } k = 1, \dots, m_v$$

- Number of design variable is $m_v = \log_2 m$
 - Possible to interpolate between $2^{(m_v-1)}$ to 2^{m_v} materials with m_v variables
- Introduction of a penalization scheme (here power law) to end-up with -1/1 values

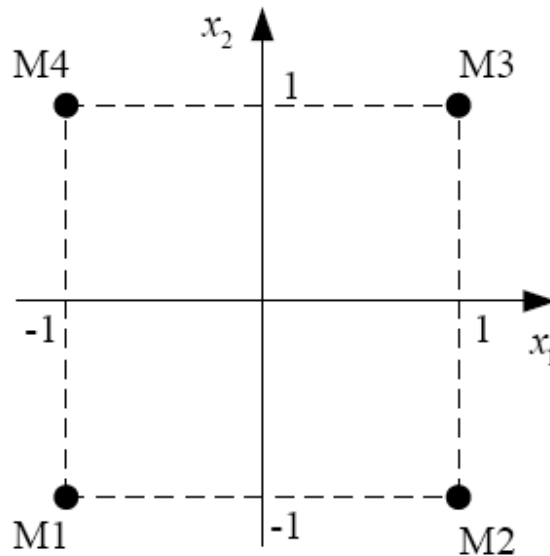


Bi-value coded parameterization

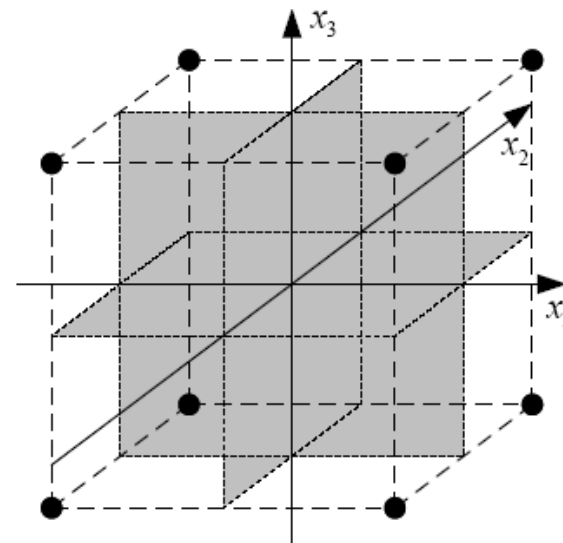
- Visualization: for $m_v=2$ and $m_v=3$, the method recovers 4-node and brick (8-node) elements shape functions.

Table 1 s_{jk} values ($m_v=2, m=4$)

$k \backslash j$	1	2	3	4
1	-1	1	1	-1
2	-1	-1	1	1



(a) $m_v=2, m=4$



(b) $m_v=3, m=8$

Penalization schemes

- To come to a solution with one single material, one introduces a penalization schemes:

- SIMP

$$f(\chi) = \chi^p$$

- RAMP (Stolpe & Svanberg, 2001)

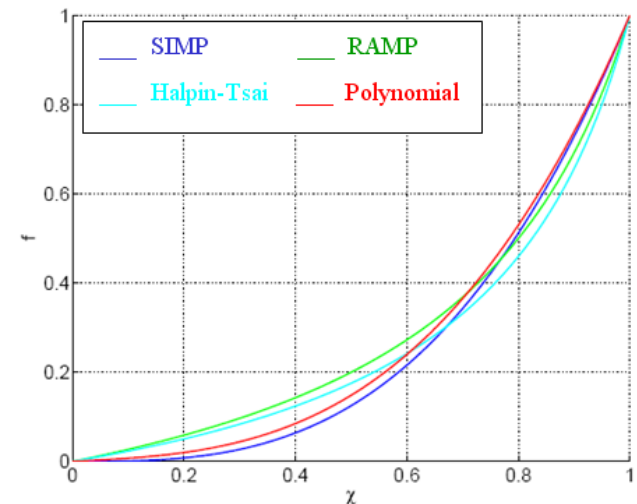
$$f(\chi) = \frac{\chi}{1 + p(1 - \chi)}$$

- Halpin Tsai (Halpin-Tsai, 1969)

$$f(\chi) = \frac{r\chi}{(1 + r) - \chi}$$

- Polynomial penalization (Zhu, 2009)

$$f(\chi) = \frac{\alpha - 1}{\alpha} \chi^p + \frac{1}{\alpha} \chi$$



Optimization Problem Formulation

- Compliance minimization under given load cases

$$\min_{\{x_{ik}\} \quad (i=1,\dots,n; k=1,\dots,m_v)} C = \mathbf{F}^T \mathbf{u}$$

$$\text{subject to: } \left(\sum_{l=1}^{n_v} \mu_l V_l \leq \overline{V} \right)$$

- For pure laminate optimization, no resource (volume) constraint is generally necessary
- Sensitivity analysis

$$\frac{\partial C}{\partial x_{ik}} = 2\mathbf{u}^T \frac{\partial \mathbf{F}}{\partial x_{ik}} - \mathbf{u}^T \frac{\partial \mathbf{K}}{\partial x_{ik}} \mathbf{u} = -\mathbf{u}^T \frac{\partial \mathbf{K}}{\partial x_{ik}} \mathbf{u}$$

- Requires the derivatives of the weighting functions

$$\frac{\partial \mathbf{K}_i}{\partial x_{ik}} = \sum_{j=1}^m \frac{\partial w_{ij}}{\partial x_{ik}} \mathbf{K}_i^{(j)}$$



Optimization Problem Formulation

- Maximization of fundamental natural frequencies

$$\begin{aligned} & \min_{\{x_{ik}\}} \omega^2 \quad (i=1, \dots, n; k=1, \dots, m_v) \\ & \text{subject to: } (\mathbf{K} - \omega^2 \mathbf{M}) \mathbf{u} = 0 \\ & -1 \leq x_{ik} \leq 1 \end{aligned}$$

$$\mathbf{M}_i = \sum_{j=1}^m v_{ij} \mathbf{M}_i^{(j)} \quad v_{ij} = \left[\frac{1}{2^{m_v}} \cdot \prod_{k=1}^{m_v} (1 + s_{ik} x_{ik}) \right]^{p_M}$$

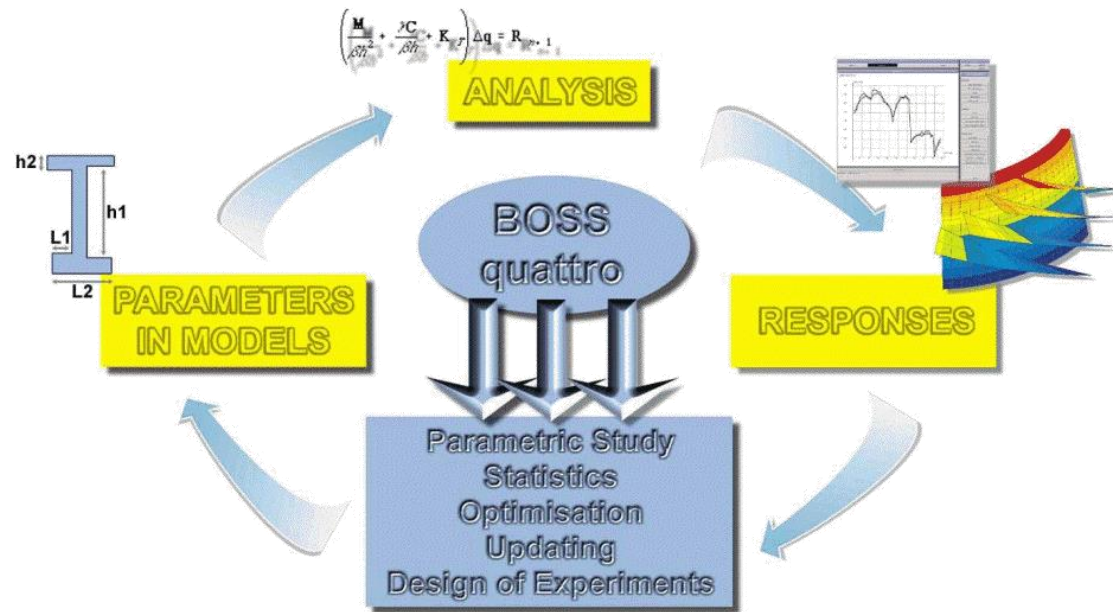
- Sensitivity analysis

$$\frac{\partial \omega^2}{\partial x_{ik}} = \frac{\mathbf{u}_i^T \frac{\partial \mathbf{K}_i}{\partial x_{ik}} \mathbf{u}_i - \omega^2 \mathbf{u}_i^T \frac{\partial \mathbf{M}_i}{\partial x_{ik}} \mathbf{u}_i}{\mathbf{u}^T \mathbf{M} \mathbf{u}}$$

- The derivatives can be either positive or negative (non monotonic function)



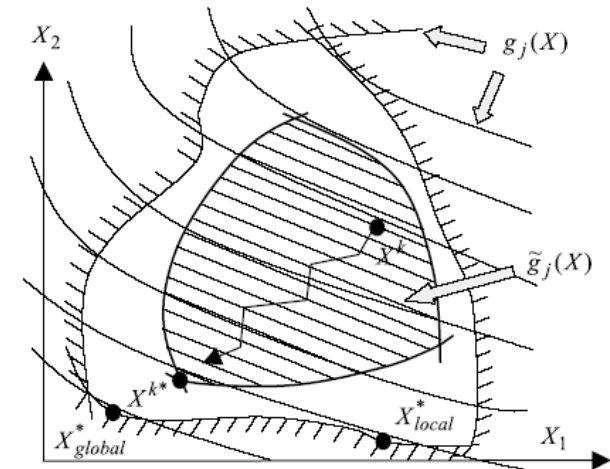
- Implementation
 - Analysis carried out in SAMCEF Composites
 - Laminate plate elements
 - Thick composite shells (8-node bricks)
 - Optimization
 - Boss Quattro Open Object Oriented platform for Optimization



Implementation & Solution procedure

■ Solution of optimization problem: Sequential Convex Programming

- Sequence of explicit subproblems
 - CONLIN (Fleury, 1989)
 - GCMMA (Bruyneel et al., 2002)
- General strategy with efficient capabilities in treating large scale Problems



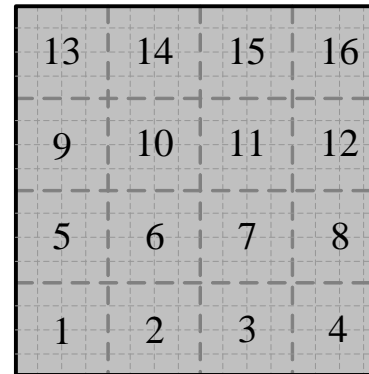
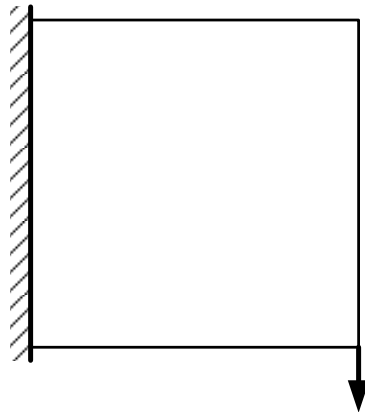
■ Remarks:

- For CONLIN variables must be >0 , so a change of variables is necessary. For instance:

$$z_i = \frac{x_i + 1}{2}$$

Numerical applications: Square plate under vertical force

- Maximum in-plane compliance problem is solved by selecting the optimal orientation of the ply



Loads and boundary conditions

Design model with 4×4 patches

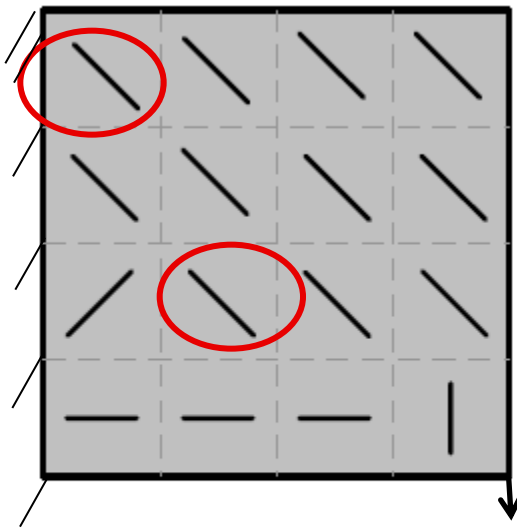
Table 4 Material properties

E_x	E_y	G_{xy}	ν_{xy}
146.86GPa	10.62GPa	5.45GPa	0.33

Table 3 Orientations

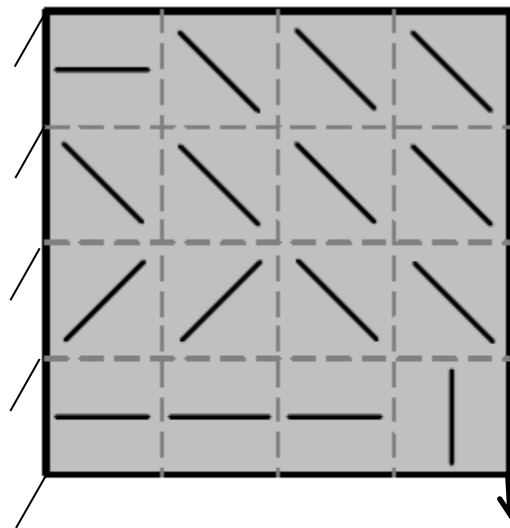
Number of material phases (m)	Number of design variables for each region (m_v)	Discrete orientation angle ($^\circ$)
4	2	90/45/0/-45
9	4	80/60/40/20/0/-20/-40/-60/-80
12	4	90/75/60/45/30/15/0/-15/-30/-45/-60/-75

Numerical applications: Square plate under vertical force



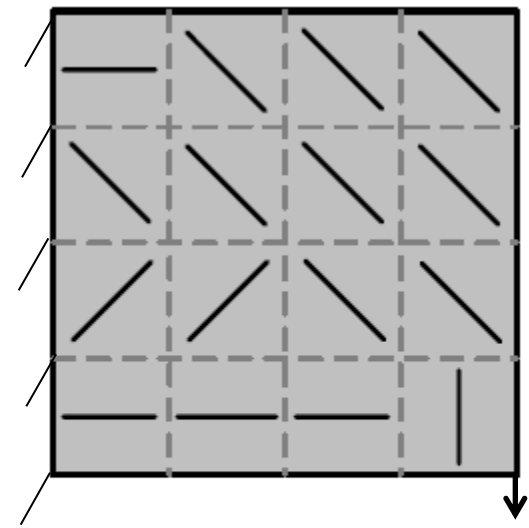
(a) DMO ($C=1.220 \times 10^{-4}$)

$m_v=4$



(b) SFP ($C=1.182 \times 10^{-4}$)

$m_v=2$



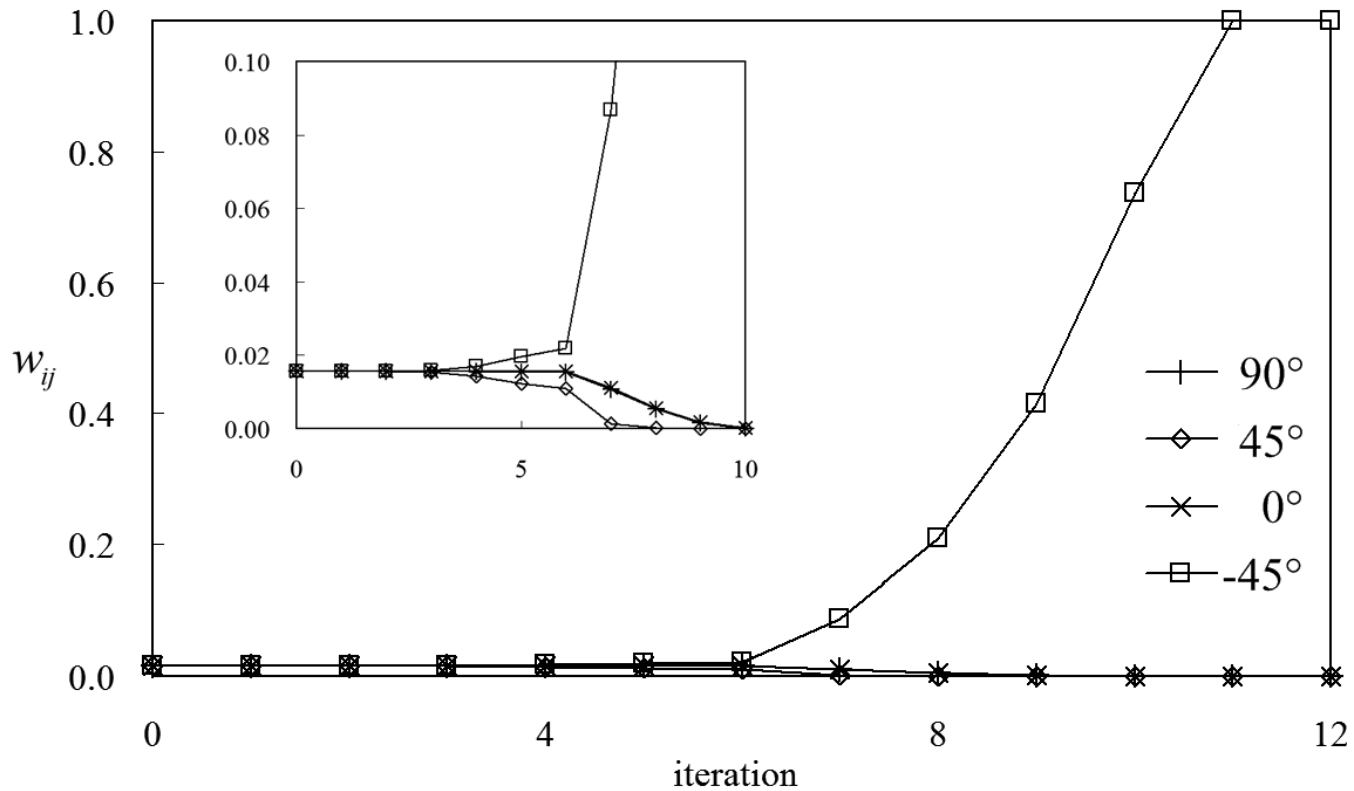
(c) BCP ($C=1.182 \times 10^{-4}$)

$m_v=2$

Optimization results of the square plate under vertical force ($m=4$)

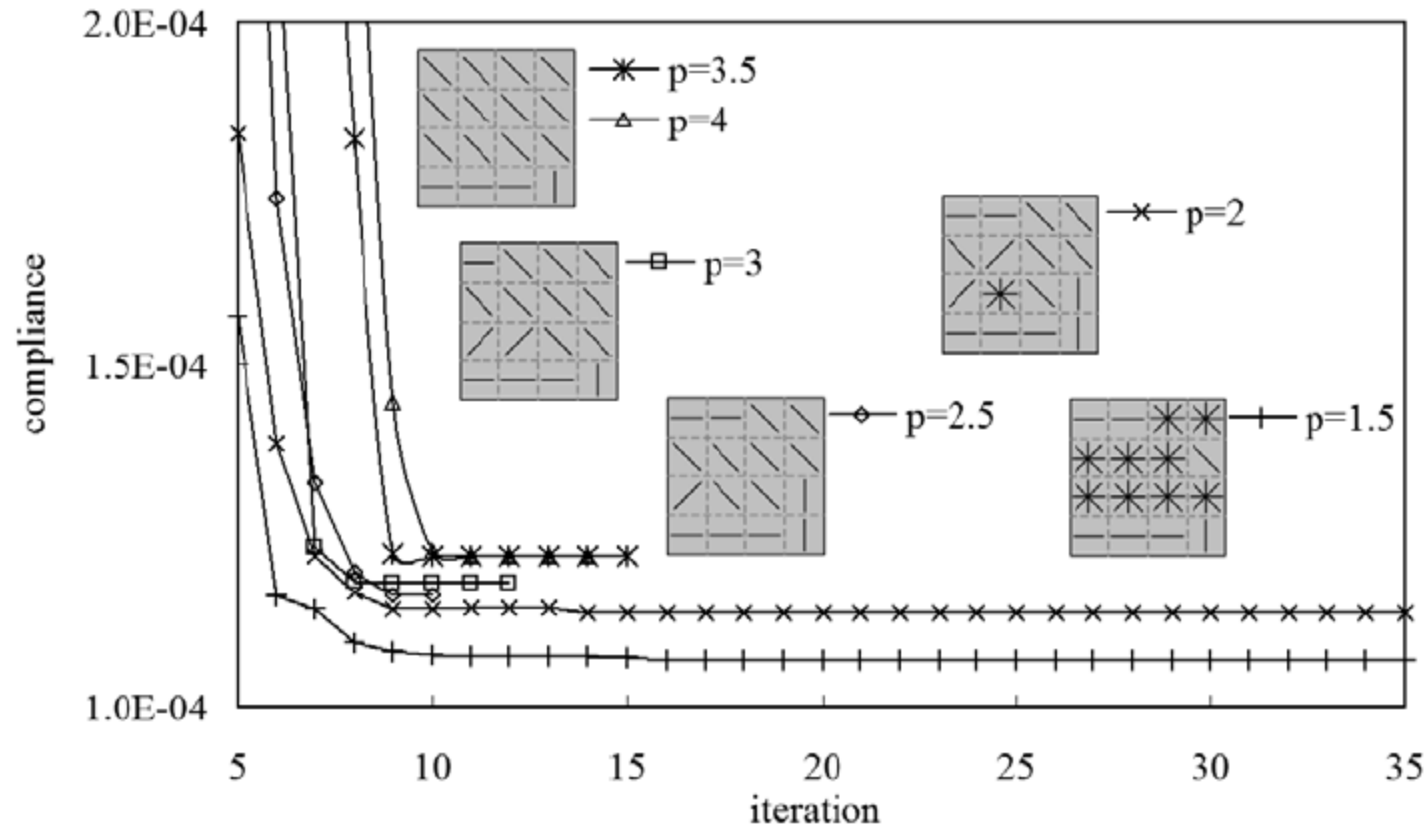


Numerical applications: Square plate under vertical force



Iteration histories of the weight for patch 16 (BCP $m=4$)

Numerical applications: Square plate under vertical force

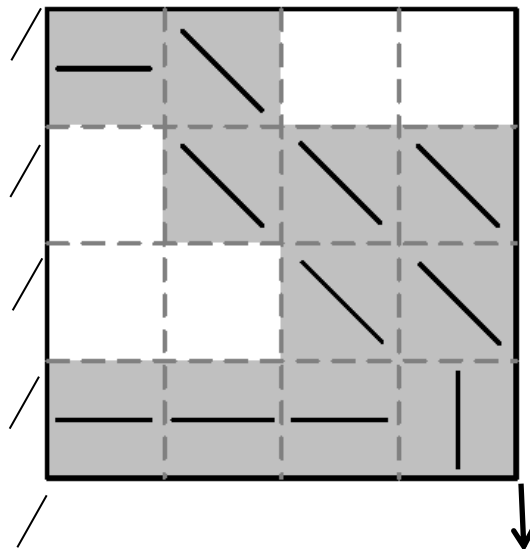


Influence of the penalization factor p of the BCP scheme upon the optimization results

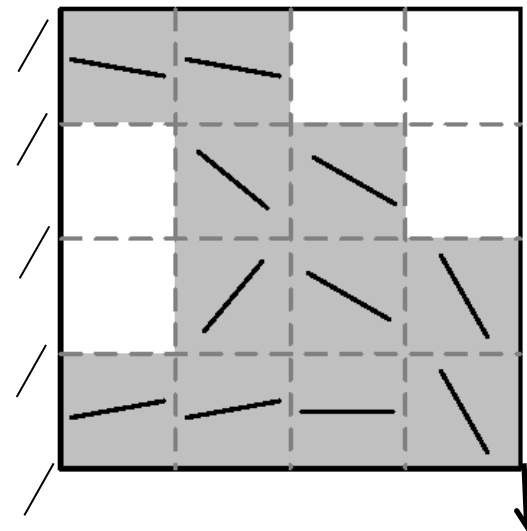


Numerical applications: Square plate under vertical force

- Topology optimization: void + laminate
- Volume constraint: $V < 11/16$



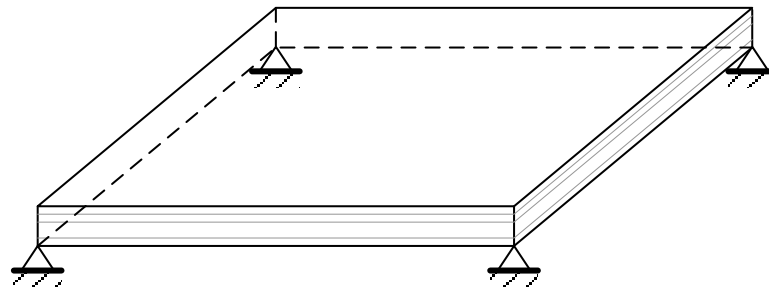
4 orientations
90/45/0/-45



18 orientations
90/80/70/60/50/40/30/20/10/0/
-10/-20/-30/-40/-50/-60/-70/-80

Numerical applications: Natural frequency maximization

- Maximum fundamental eigenfrequency of a square plate supported at its four corners

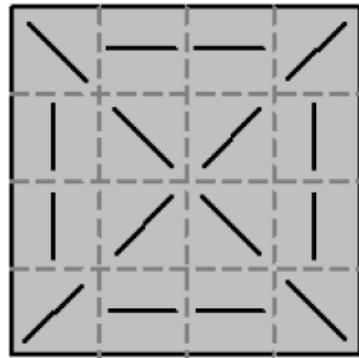


4 candidate orientations (90/45/0/-45)

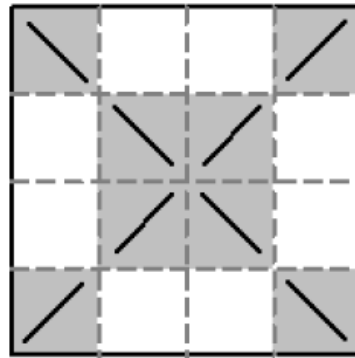
Table 1 Material properties (MC1)

E_x	E_y	G_{xy}	ν_{xy}
146.86GPa	10.62GPa	5.45GPa	0.33

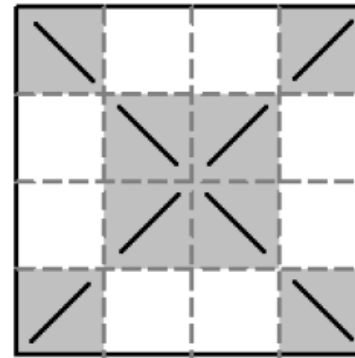
Numerical applications: Natural frequency maximization



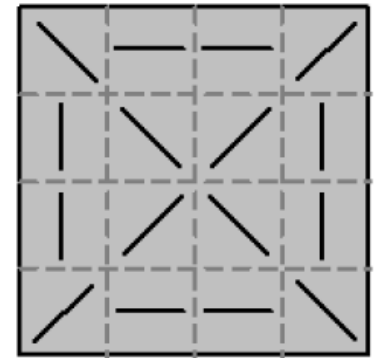
Layer 1



Layer 2

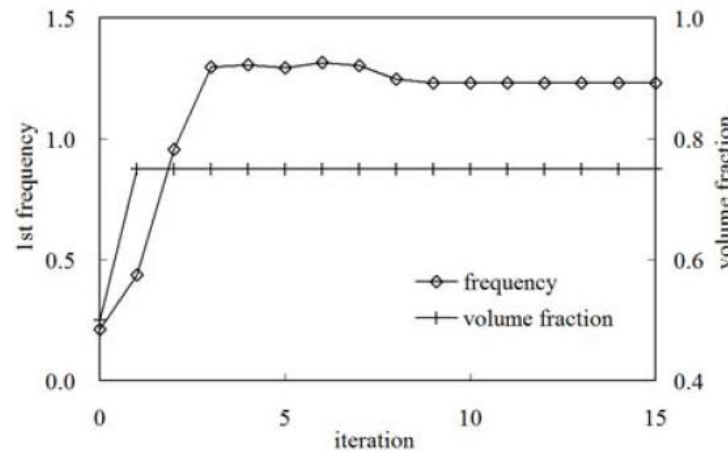


Layer 3

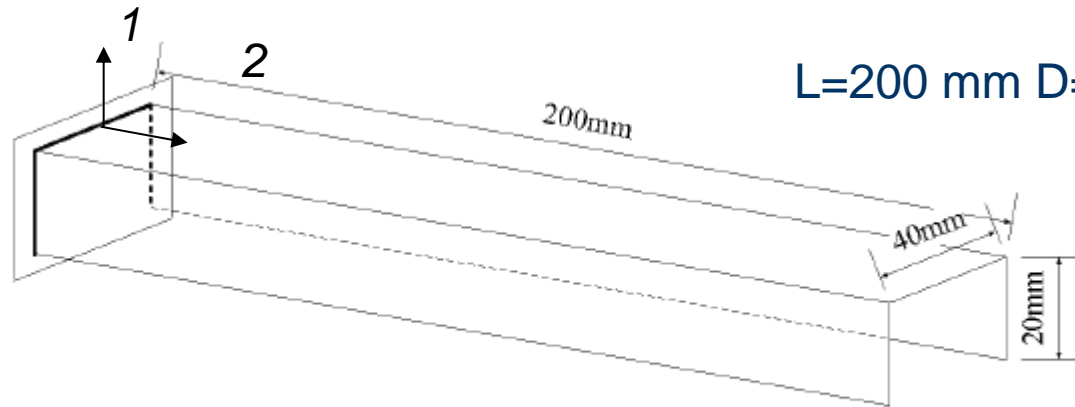


Layer 4

*Optimization results of the square plate with maximum eigenfrequency
Volume composite material < 75%*



Numerical application: long box

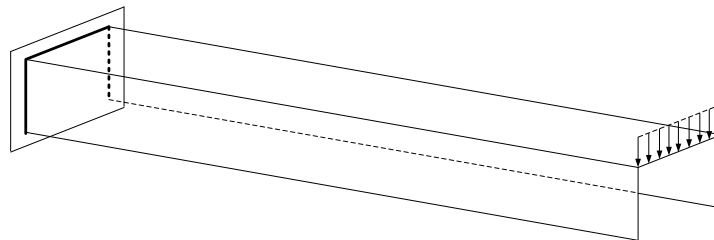


$L=200 \text{ mm}$ $D=40 \text{ mm}$ $T=1 \text{ mm}$

Fig.10 Model of a 4-layer laminated beam

Element size 4x4 mm

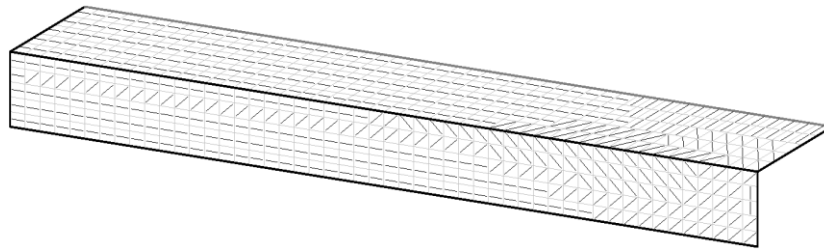
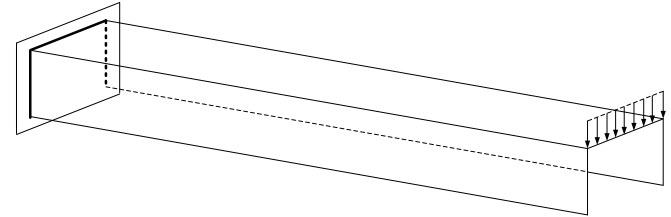
- 4 layers; element size=4
- Orientation: 90/45/0/-45
in 1-2 plane for each element axes
- Load case: line load at the tip



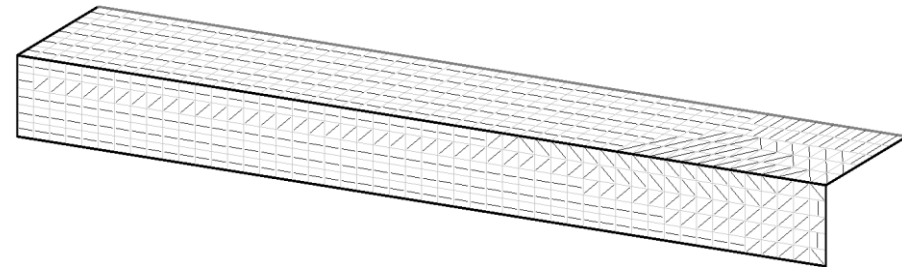
Numerical application: long box

Line force

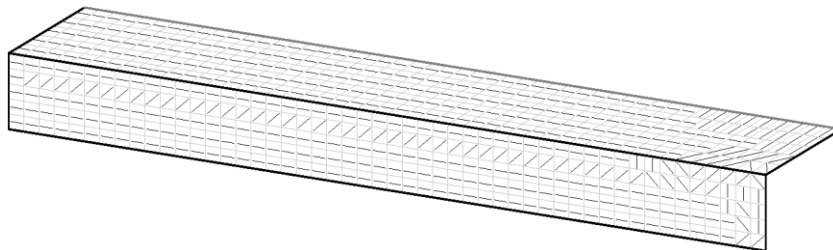
- Objective: minimize compliance
- 90° / 45° / 0° / -45°



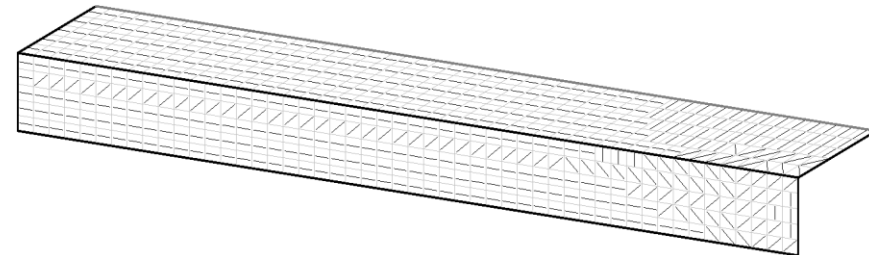
Layer 1 (inner ply)



Layer 2



Layer 3

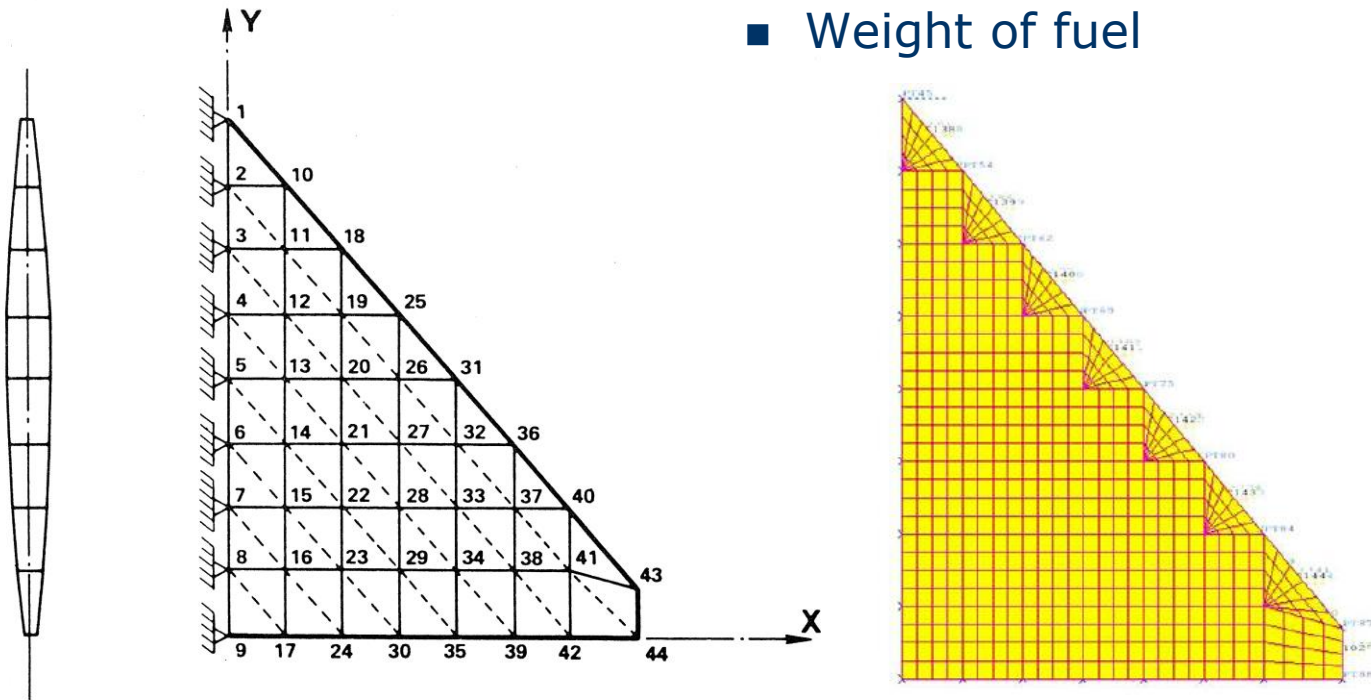


Layer 4 (outer ply)

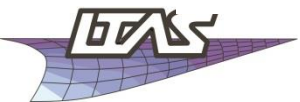


Numerical application: delta wing

- Revisit classic applications involving composite structures
 - Two load cases: Upside and downside pressure distributions
 - Weight of fuel

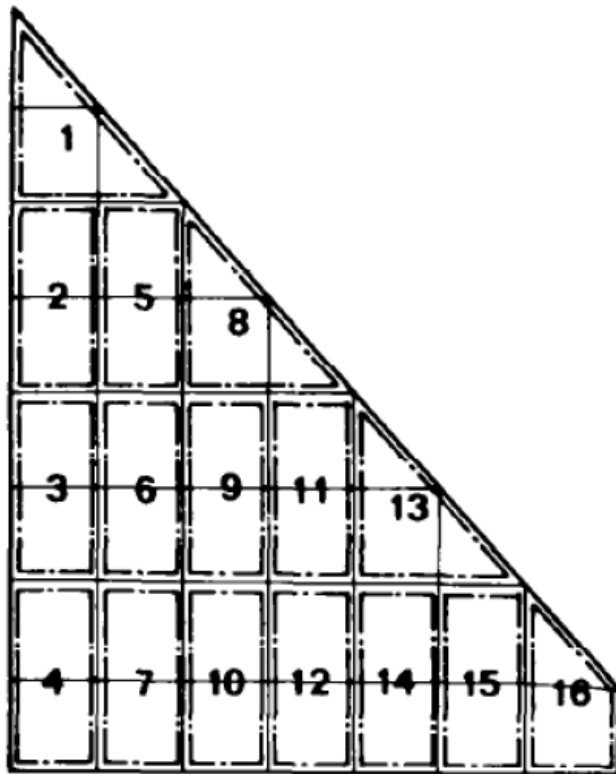


Schmidt and Fleury (1980) Dual and Approximation Concepts in Structural Synthesis. NASA Contractor Report 3226. Dec 1980.

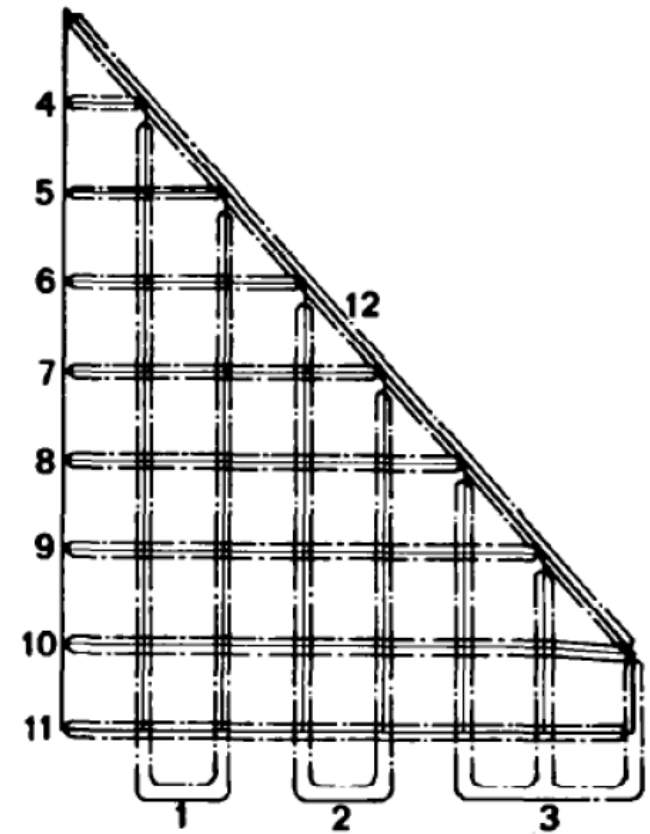


Numerical application: delta wing

■ Definition of design variables



16 design composite panels
(0° , 45° , 90° , -45°)

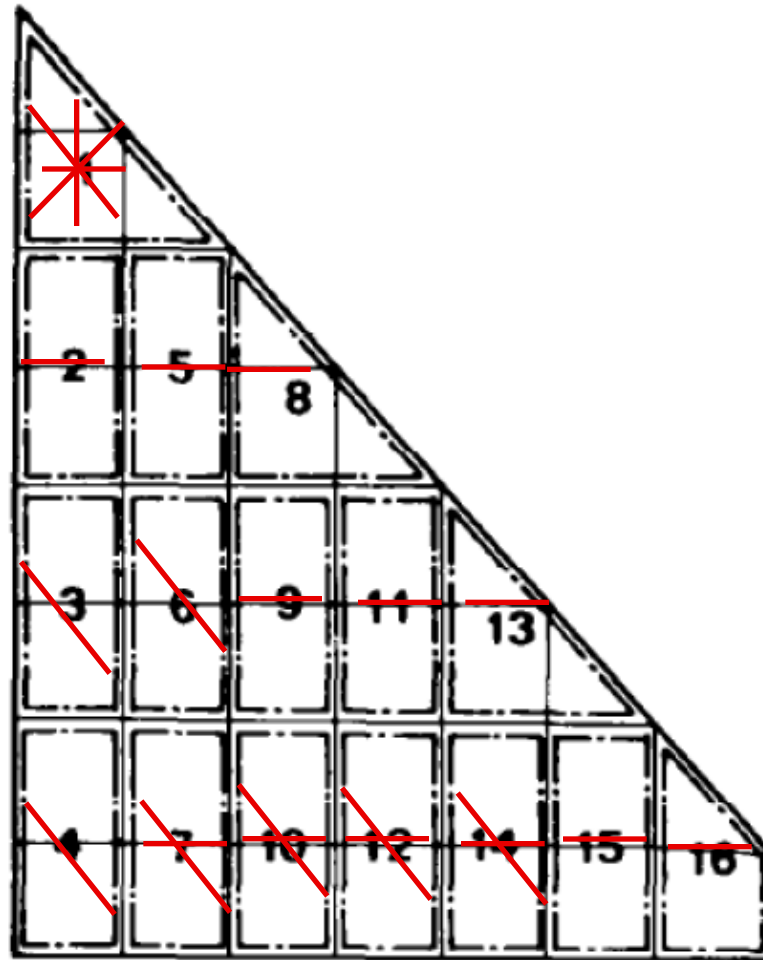


12 metallic spars and webs



Numerical application: delta wing

- Optimal design (31 iterations)



CONCLUSIONS

- Discrete Material Optimization is an interesting and alternative approach for laminate & composite structure optimization
 - For optimal layout of composite laminates over the structure
 - For stacking sequence of composite panels
- Comparison of different interpolation schemes
 - Pioneer work by Stegmann and Lund (2005)
 - Several interpolation schemes (DMO1...5)
 - New approach by Bruyneel (2011) with the Shape Function with Parameterization (SFP)
 - Limited to four materials (0° / 90° / -45° / 45°) or three materials (0° / 90° / (45° / -45°))
 - Generalization using Bi-value Coding Parameterization (BCP) (Gao et al. 2012)



CONCLUSIONS

- SFP and Bi-value Coding Parameterization mitigate the dramatic increase of design variables of DMO approach
- BCP formulation is suited for a quite efficient solution using sequential convex programming algorithms (15-30 iterations necessary)
- DMO is validated on academic applications



ON GOING WORK and PERSPECTIVES

- Address simultaneously the in-plane and the stacking sequence problems
- Extend the scope of the approach
 - Displacements
 - Stress constraints (Tsai Wu, Puck, etc.)
 - Buckling constraint
 - Non linear analysis (non linear buckling)
- Extend the application of BCP/SFP parameterization schemes to larger problems involving industrial composite structures
- Develop pre / post CAE tools to ease the data introduction and the visualization of results



THANK YOU VERY MUCH FOR YOUR ATTENTION



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